Amendments to the Specification:

Please replace paragraph [0026] with the following amended paragraph:

[0026] The embodiment shown in Figure 1 of the adjusting device 26 is characterized by a cylinder-piston unit 36, comprising a cylinder 32, in which a piston 34 coupled to the mechanical built-in parts 3 is guided. The cylinder-piston unit 36 comprises two working chambers, a first working chamber 25, in which a front side 33 of the piston 34 is subjected to pressure by the pressure media from an intermediate chamber 30 between the inside circumference 28 of the impeller pan 18 and the impeller 6, which is coupled to the chamber 19, and an additional working chamber 35, which impinges the piston 34 on a front side 37 turned away from the first working chamber 25. Further a spring device 38 is provided which brings the piston 34 to a predefined starting position and with this also to the mechanical built-in units 3 coupled thereto. The second working chamber 35 is additionally coupled to the control pressure media supply system 67. This system can for example be formed by a space in the environment of the starting unit such as the transmission. A valve device 40 is provided for setting the differential pressure, said valve device which can be clocked. In the described case, the mechanical built-in units 3 are connected to the working piston 34 in respect of the direction of its movement. In the process the movement takes place in such a way that in the starting range, i.e. range of high slip (low torque transmission) the influence is given by the mechanical built-in units and in the region of low slip this influence is reduced or completely eliminated. The placement of the mechanical built-in units is therefore set as a function of the pressures, in particular on the individual built-in unit. In the starting region of the hydrodynamic clutch the wall region 43 is completely moved and is moved during the power transfer via the hydrodynamic clutch back in the region of low slip (high torque transmission), so that it again assumes the guiding function for the flow circuit.

[0028] Figure 2a illustrates a first embodiment option of a starting unit 1.2 with means 2.2 for influencing the transmission behavior in the form of so-called annular slide valves 27, which become active in the working chamber 8. Said annular slide valves are partially annular elements which extend in axial direction at least partially through a turbine wheel whereby the surface of said slide valves characterizing the inside circumference is aligned at

an angle to the flow circuit in the working chamber 8, preferably near vertical. The annular slide valve 27 is assigned to the primary turbine wheel 6 here. Said slide valve is guided on the primary turbine wheel 6 and can be moved opposite said primary turbine wheel in axial direction, i.e. parallel to the rotational axis R. The movement consequently takes place contrary to the embodiment in Figure 1 in the range of very high slip (low torque transmission) into the working chamber 8 and in the non-influencing state (high torque transmission) into the intermediate space 30 formed between the inside circumference 28 of the housing 17 or the impeller pan 18 and the outside circumference 29 of the impeller 6.2, whereby then either no influence or only a slight influence of the flow circuit is given in the working chamber 8. An adjusting device 31 is assigned to the annular slide valve 27. Said adjusting device is the adjusting device 26 in accordance with Figure 1. This is preferably supported on the housing 17, in the represented case of the impeller pan 18, and coupled to the annular slide valve 27. The adjusting device 31 comprises a cylinder-piston unit 36 which can be activated by pressure media, comprising a cylinder 32, whose working piston 34 is connected to the annular slide valve 27. The front side 33 coupled to the annular slide valve 27 can be subjected to the action of pressure by pressure media from the interior 30, which is coupled to operating means supply channel or space 19. The other front side of the piston 34 can be subjected to the action of pressure by the control pressure from the control pressure media source 67. The cylinder 32 can in the process be formed by the housing 17 or the impeller pan 18 or an element that is connected thereto in a stationary or rotationally fixed manner. The working piston 34 is then guided in this cylinder. The coupling of the first working chamber 25 to the operating means supply channel or space 19 or the operating means supply channel 20 takes place via the intermediate space 30 connected thereto. The connection of the operating means supply channel or space 19 to the operating means supply line 20 takes place for example via a line integrated into the housing and/or a channel and/or space. Preferably the formation of the channels takes place by integration in wall regions on the impeller pan 18, separate pipelines are also conceivable. The arrangement of the adjusting device occurs preferable in a region which lies in axial direction in the direction of movement of the annular slide valve 27. In the embodiment represented in Figure 2a the arrange takes place in the process outside of the impeller pan 18. Extensions into the intermediate space 30 are also conceivable and are represented in Figure 2b. Further it is conceivable to completely

integrate this cylinder-piston unit 36 in the housing 17 in accordance with Figure 2c, whereby also here attention is paid to the connection to the piston 34, so that the coupling to the mechanical built-in parts preferably always lies at the front side pointing contrary to the movement of direction between the desired position in the case of high slip and low slip and consequently the subjection to the action of pressure from the operating means supply chamber or space 19 always takes place in the direction of movement from the neutral position, i.e. a position in the non-influencing state to the influencing state. The connection of the mechanical built-in parts 3 to the piston 34 in the direction of movement is also conceivable. However, in this case the pressures are to be selected correspondingly.